


CALCULATION SHEET 	Document Ref:	<i>SX042a-EN-EU</i>	Sheet	<i>1</i>	of	<i>5</i>
	Title	<i>Example: Parametric fire curve for a fire compartment</i>				
	Eurocode Ref	<i>EN 1991-1-2:2002</i>				
	Made by	<i>Z Sokol</i>	Date	<i>Jan 2006</i>		
	Checked by	<i>F Wald</i>	Date	<i>Jan 2006</i>		

Example: Parametric fire curve for a fire compartment

This example shows the determination of the parametric fire curve for a fire compartment in an office building, according to Annex A of EN 1991-1-2. The walls and the floors above and below the compartment are of reinforced concrete; the walls have several openings. Reference is made to SD006 for the thermal properties of the walls and floors.

Basic data

Dimensions of the fire compartment

width: $a = 8,5$ m
length: $b = 10,0$ m
height: $h = 3,15$ m
height of opening $h_{op} = 1,537$ m
width of opening $b_{op} = 3,85$ m
number of openings $n = 4$; see **Figure 1**.

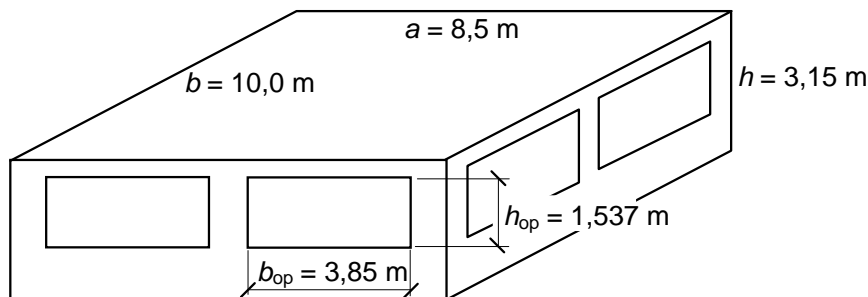


Figure 1: Considered fire compartment

The floor and ceiling are made from reinforced concrete


density $\rho = 2300$ kg m⁻³
specific heat $c = 840$ J kg⁻¹ K⁻¹
thermal conductivity $\lambda = 1,57$ W m⁻¹ K⁻¹


[SD006](#)

The walls are made from lightweight concrete

density $\rho = 500$ kg m⁻³
specific heat $c = 840$ J kg⁻¹ K⁻¹
thermal conductivity $\lambda = 0,22$ W m⁻¹ K⁻¹

[SD006](#)

CALCULATION SHEET 	Document Ref:	<i>SX042a-EN-EU</i>	Sheet	2	of	5	
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<p><u>Fire load density</u></p> <p>For office buildings, the characteristic fire load density related to floor area, for the 80% fractile case (Gumbel distribution) is given by Table E.4 as:</p> $q_{f,k} = 511 \text{ MJ m}^{-2}$ <p>The floor area is</p> $A_f = a \cdot b = 8,5 \cdot 10,0 = 85 \text{ m}^2$ <p>The factor to take into account the fire activation risk due to the size of the compartment is given by linear interpolation from Table E.1:</p> $\delta_{q1} = 1,1 + (1,5 + 1,1) \cdot (85 - 25) / (250 - 25) = 1,20$ <p>For the factor to take into account the fire activation risk due to the type of occupancy $\delta_{q2} = 1,00$.</p> <p>The factor to take into account the different fire fighting measures $\delta_n = 1,00$</p> <p>The design fire load density is given by:</p> $q_{f,d} = q_{f,k} \delta_{q1} \delta_{q2} \delta_n$ $= 511 \cdot 1,20 \cdot 1,00 \cdot 1,00 = 613 \text{ MJ m}^{-2}$ <p><u>Thermal properties of the fire compartment</u></p> <p>The total area of the enclosure is:</p> $A_t = 2 A_f + 2 (a + b) h = 2 \cdot 85 + 2 \cdot (8,5 + 10,0) \cdot 3,15 = 286,55 \text{ m}^2$ <p>The total area of vertical openings is:</p> $A_v = n h_{op} b_{op} = 4 \cdot 1,537 \cdot 3,85 = 23,67 \text{ m}^2$ <p>The surface factor for floor and ceiling is:</p> $b = \sqrt{\rho c \lambda} = \sqrt{2300 \cdot 840 \cdot 1,57} = 1742 \text{ J m}^{-2} \text{ s}^{-0,5} \text{ K}^{-1}$ <p>The surface factor for walls is:</p> $b = \sqrt{\rho c \lambda} = \sqrt{500 \cdot 840 \cdot 0,22} = 304 \text{ J m}^{-2} \text{ s}^{-0,5} \text{ K}^{-1}$ <p>Both values are within the limit $100 \leq b \leq 2200$.</p>							<p>EN 1991-1-2 Annex E Table E.4</p> <p>EN 1991-1-2 Annex E Table E.1</p> <p>EN 1991-1-2 Annex A</p>

CALCULATION SHEET 	Document Ref:	<i>SX042a-EN-EU</i>	Sheet	3	of	5
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The resulting surface factor is

$$b = \frac{\sum(b_i A_i)}{A_t - A_v} = \frac{2 \cdot 85 \cdot 7 \cdot 1742 + (2 \cdot (8,5 + 10,0) \cdot 3,15 - 23,67) \cdot 304}{286,55 - 23,67} = 1234 \text{ J m}^{-2} \text{ s}^{-0,5} \text{ K}^{-1}$$

Ventilation properties of the fire compartment

The opening factor is:

$$O = \frac{A_v \cdot \sqrt{h_{eq}}}{A_t} = \frac{23,67 \cdot \sqrt{1,537}}{286,55} = 0,1024 \text{ m}^{0,5}$$

where the weighted average height of the openings $h_{eq} = 1,537 \text{ m}$.

The opening factor should be within the limits $0,02 \leq O \leq 0,2 \text{ [m}^{0,5}]$. The limitation is satisfied.

Time factor function

The Γ factor is given by:


$$\Gamma = \frac{\left(\frac{O}{b}\right)^2}{\left(\frac{0,04}{1160}\right)^2} = \frac{\left(\frac{0,1024}{1234}\right)^2}{\left(\frac{0,04}{1160}\right)^2} = 5,791$$

Fire load density related to surface area

The design fire load density related to the surface area is given by:

$$q_{t,d} = \frac{q_{f,d} A_f}{A_t} = \frac{613 \cdot 85}{286,55} = 181,8 \text{ MJ m}^{-2}$$

EN 1991-1-2
[Annex A\(3\)](#)

CALCULATION SHEET 	Document Ref:	<i>SX042a-EN-EU</i>	Sheet	<i>4</i> of <i>5</i>
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Evaluation of the limiting time and maximum temperature

Medium fire growth rate is expected, with $t_{lim} = 20 \text{ min} = 0,333 \text{ hour}$.

The time t_{max} to reach the maximum temperature is given by:

$$t_{max} = \max \left\{ \frac{0,2 \cdot 10^{-3} q_{t,d}}{O}, t_{lim} \right\} = \max \left\{ \frac{0,2 \cdot 10^{-3} \cdot 181,8}{0,1024}, 0,333 \right\} = 0,355 \text{ hour}$$

The fire is ventilation-controlled. because t_{max} is given by the first term.

The time to reach maximum temperature, taking account of the openings and thermal absorptivity, t_{max}^* is given by:

$$t_{max}^* = t_{max} \Gamma = 0,355 \cdot 5,791 = 2,056 \text{ hour}$$

and the maximum gas temperature is given by:

$$\theta_{max} = 20 + 1325 \left(1 - 0,324 e^{-0,2 \cdot 2,056} - 0,204 e^{-1,7 \cdot 2,056} - 0,472 e^{-19 \cdot 2,056} \right) = 1052^\circ\text{C}$$

The curve in the heating phase

The gas temperature in the heating phase is given by

$$\theta_{g,t} = 20 + 1325 \left(1 - 0,324 e^{-0,2 t^*} - 0,204 e^{-1,7 t^*} - 0,472 e^{-19 t^*} \right)$$

where the time t^* is given by:

$$t^* = t \Gamma = 5,791 t$$

The curve in the cooling phase

For $t_{max}^* > 2$ hours, the gas temperature in the cooling phase is given by:

$$\begin{aligned} \theta_{g,t} &= \theta_{max} - 250 \left(t^* - t_{max}^* x \right) = \\ &= 1052 - 250 \cdot \left(t^* - 2,056 \cdot 1 \right) = \\ &= 1566 - 250 \cdot t^* \end{aligned}$$

where the factor $x = 1$ for a ventilation-controlled fire.

The resulting parametric curve is shown in **Figure 2**.

CALCULATION SHEET



Document Ref:	<i>SX042a-EN-EU</i>	Sheet	<i>5</i>	of	<i>5</i>
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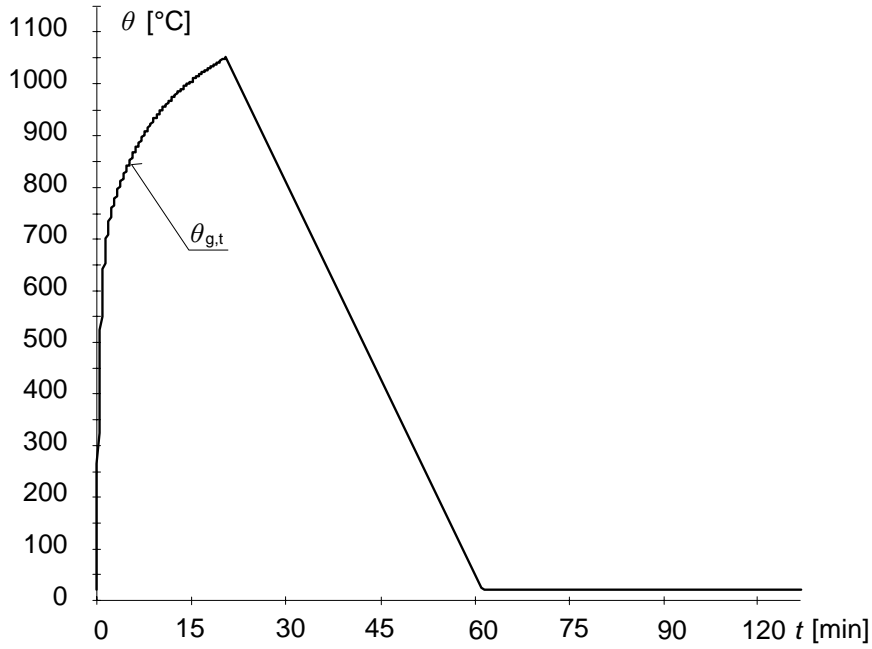


Figure 2: Gas temperature-time curve



Quality Record

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